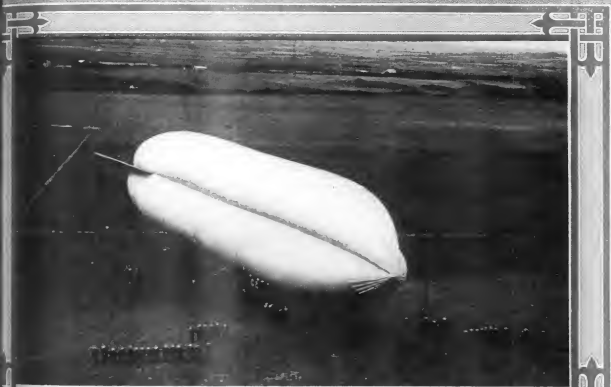


AVIATION AND AERONAUTICAL ENGINEERING



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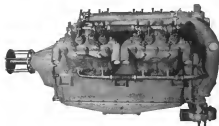


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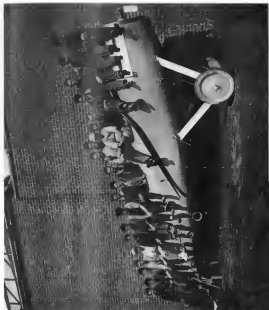


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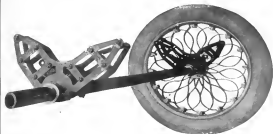


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INDEX TO CONTENTS

	PAGE		PAGE
Working Table and Method of Testing Model	455	Penology Up and Balancing Airplane Propellers	462
Aircrosses at N. P. L.	456	The Future of Commercial Aeronautics	464
Description of the Albatross Engine	456	International Aircraft Standards	466
File Dried Airplane Woods	454	Digest of the Foreign Aeronautical Press	473
Book Reviews	461	News of the Forefront	475

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No. 1

November 1, 1937

No. 1

Whirling Table and Method of Testing Model Airscrews at N. P. L.

By A. Faye, A.R.C.S., D.L.C.

At the present time there are two different methods of testing airscrews at the National Physical Laboratory, (a) with an airscrew dynamometer mounted on a whirling table and, (b) with an airscrew balance designed for use in a wind tunnel.

The present article deals only with the first of these methods.

of the whirling table is carried up from the floor to the roof, the arm being supported by a system of levers for a counter-balance built up of light angle iron and provided with cross bars and steel wire ties. The whirling table is driven by a 15 hp electric motor, through a worm reduction gear of 20 to 1.



FIG. 1. WHIRLING TABLE AT THE N. P. L.

able, but the author hopes to obtain permission to publish, in the near future, a description of the new screw balance.

Description of the Whirling Table and the Aiscrew Dynamometer

A brief description of the whirling table as first designed is given in the "Evolution Report of the Advisory Committee for Aerodynamics, Year 1920-1921," but more than time both the dynamometer and the method of experimenting have been considerably modified and improved.

The whirling table, a photograph of which is given in Fig. 1, is mounted on a perforated steel stand, 80 ft. by 80 ft., so that the airscrew experiments may be carried on independently of the atmospheric conditions. The over all diameter of the whirling table is 40 ft., the dynamometer being situated at the end of an arm of length 30 ft., which is built up of light steel tubes tapering from 1½ in. diameter at the base to 1 in. diameter at the extremity. These tubes are spaced 12½ in. apart and are connected together by struts. The central post

is fixed to the central post by the inertia of the arm when the motor is stopped, the post is not immediately above the worm wheel and the upper and lower parts connected by a pinion gear which allows the free rotation of the arm.

The speed of rotation of the whirling table, which is regulated by an adjustable resistance in the armature circuit of the motor, can be varied from 3 to 17 rpm., corresponding with translational speeds at the end of the arm of 0 to 35 m.p.h.

For the purpose of transmitting current to the motor during the rotation, and also in the measuring apparatus, during the rotation of the whirling table a set of eight slip rings are fixed to the post.

The motor shaft is driven by a rubber roller belt by a 15 hp. motor mounted on the arm, the speed of the motor being regulated from an observation table in the corner of the whirling shed.

A small resistance counter driven by the motor which is connected electrically so as to ring a bell every hundred rev-

* See the latest reports of the Advisory Committee for Aerodynamics.

lettern, the airscrew speed being timed with a stop-watch for a period of about two seconds. A second bell rings at each revolution of the whirling table, so that by timing the stopwatch against a stop-watch the speed of rotation of the whirling

table and of air pressure due to the centrifugal head in the conveying pipe the reading of the dynamometer in proportion to $V \rightarrow V^2 \rightarrow P$ that $Q_0, V^2 = 2gH$, from which the velocity of the swirl may be calculated since V is a measured quantity. The

FIG. 2. PROPELLER DYNAMOMETER—GENERAL ARRANGEMENT

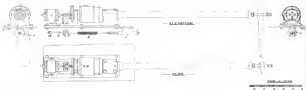


table and hence the speed of translation of the airscrew due to motion of the whirling table may be calculated.

As would be expected, the air in the whirling table is not in motion by the whirling table and also by the backward of the airscrew, the velocity of translation of the airscrew through the air being obtained from the measurements of both the velocity of translation of the airscrew due to the rotation of the whirling table and the velocity of swirl of the air. The velocity of the air swirl is measured by a Pitot tube mounted at the end of a light arm opposite to that carrying the airscrew, the tube following the direction of the air at a distance of about 100 ft. so that it is well out of the region of the direct action of the

speed of the air swirl, when the airscrew is not working, is about 8 per cent of the true speed.

A series of measurements of the air swirl is made at each determination of the performance of the airscrew upon the swirl is locally depends on both the rotation speed of the whirling table and the slip stream at which the airscrew is working. At a low translational speed when the airscrew is going to thrust the velocity of the air swirl may be negative.

Description of the Dynamometer

Two sketches of the dynamometer, a general arrangement and a sectional drawing, are given in Figs. 2 and 3.

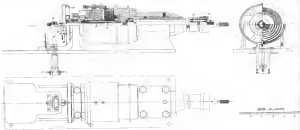


FIG. 3. PROPELLER DYNAMOMETER—SECTION

slip stream. The tube is mounted through an airtight masonry seal at the top of the central post to one limb of a manometer, the other limb being open to the atmosphere.

If V be the velocity at the end of the whirling arm and v the velocity of the air swirl in the swirl, the velocity of the tube relative to the air is $V-v$, so that taking into consideration the

The dynamometer was designed to measure a maximum thrust of 15 lb and a maximum torque of 4 lb./ft., but with an airscrew of 5 ft. made measurements greater than 4 lb. thrust and 1 lb./ft. for the torque are easily made.

The airscrew shaft is driven through two coiled springs, the extension of the springs, which is proportional to the

torque, being measured by the relative displacement of a pointer over a small drum α , which carries a small slip of paper. The position of the pointer at any time during the experiment is recorded by passing a series of sparks through the paper, so that if the zero position of the pointer is also



FIG. 4. DIAGRAMMATIC SKETCH OF TORQUE BALANCING APPARATUS

recorded on the paper the magnitude of the torque may be directly calculated from the known calibration of the springs.

Adjustable torque springs are wound in opposite directions so that tendencies to rotate due to centrifugal force along counterbalance each other. A small torque correction, the magnitude of which is a function of the rotational speed, is made to each torque measurement.

The airscrew shaft is allowed a small axial movement, about 1.200 in., by the aid of the ball bearings β, β', β'' , which have parallel outer races. The end of the airscrew shaft bears

against an oscillating lever d which is pivoted at its center, the lower end of the lever being directly actuated by the thrust spring b , the tension in which is adjusted by the peep hole f , and two adjustable stops c, c' , both stops being insulated electrically from the rest of the dynamometer.

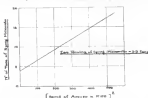


FIG. 5. RELATION OF POWER HEAD TO POWER HEAD SQUARED IN VARIOUS CASES

The method of mounting when the thrust of the airscrew is balanced by the pull of the spring is illustrated by the diagrammatic sketch of Fig. 4. From the diagram it is seen that the direction of the current through the galvanometer depends whether the lever is touching stop c or stop c' . When the thrust of the airscrew balances the pull of the thrust spring, and the oscillating lever is passing midway between the two stops, currents of equal magnitude but of opposite direction flow through the galvanometer so that the galvanometer needle is not deflected owing to the inertia of the moving parts of the

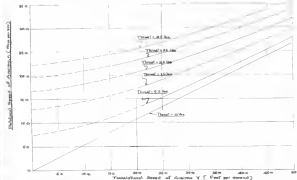


FIG. 6. RELATION BETWEEN TRANSLATIONAL AND ROTATIONAL SPEEDS AT A CONSTANT THRUST OF AN AIRSCREW

gyrometer and the indicators of the current.

The accuracy of this method of balancing is very great, and as the distance between the aerometer is not affected by any variation of the battery voltage. Moreover, any variation of the current resistance at either spot is very small compared with the large resistance of the current. Each spot has a hard-sold steel contact which makes contact with a small flat hard-sold steel plate at the bottom of the coil winding.

Observations in the torque springs due to air small variability of the driving torque are deemed considerable by the

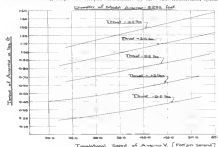


FIG. 7. RELATIONSHIP BETWEEN TRANSLATIONAL SPEED AND TORQUE OF GYROMETER TORQUE OF TORQUE

oil dash pot K , in Fig. 4. The dash pot consists of a series of concentric tubes, all inside tubes being attached to a sleeve carrying the inner end of the torque section, and the outer end to the motor of which are fixed the outer ends of the springs. The resistance of one end of a torque spring relative to the other are dampened out by the viscosity of the oil between the two series of tubes.

The plane of the air screw passes through the axis of rotation of the rotating table, and the centrifugal force due to the weight of the air screw has no component in the direction of the thrust. The centrifugal force acting on the air screw should have a small component in the direction of the thrust which is satisfactorily accounted for by the centrifugal force of a small mass of string threaded a hole in each lever s , Fig. 3.

The Method of Conducting the Air Screw Experiments

It is now proposed to describe the method of conducting an air screw experiment, the article being illustrated with the data taken from experiments on a model air screw. The scale of the model air screw was 1/144 size, the diameter of the full-size air screw being 9 ft. 2 in.

Firstly, the reading of the project head f when there is no torque in the thrust spring is obtained from the static experiments on the air screw, namely, experiments made with the air screw rotating at a fixed speed. In Fig. 5 the reading of the project head has been plotted against the square of the rotational speed. The reading of the project head, when the air screw is not rotating and also when giving no thrust, is 2.6 inches, as that from the calculation of the thrust spring—3 times represent 1 lb.—the thrust of the air screw at any reading of the project head is known.

The testing of an air screw under working conditions similar to those of positive is now commenced. The thrust spring is

given a tension of known magnitude and the rotating table is slowly speed of rotation, and then the speed of rotation of the air screw is adjusted until the thrust of the air screw balances the tension in the thrust spring. The distance between the project head and the indicator is then measured. The air screw is then rotated for a period of about ten minutes, during which time the translational speed at the end of the rotating arm, the rotational speed of the air screw, and the velocity of wind of the air are all measured. At the termination of each series of experiments, the tension of the thrust spring is changed, and the measurements of the rotating and translational speeds of the air screw are being made, the torque is electrically measured.

The experiments are conducted by two observers, one observing the indicator of the thrust balance apparatus, recording the torque and measuring the rotational speed of the air screw, the other using the measurements of the rotational speed of the arm and the air velocity at the air. Usually twenty sets of observations are needed to completely define the performance of the air screw, each set taking about 4 min.

The data of the experiments made with the model air screw tested in this instance are given in Figs. 6 and 7. The curves of Fig. 6 show the relationship between the translational and rotational speeds of the air screw at constant values of the thrust, from which it will be noted that the experimental results—that is, the distance moved forward into the air during one revolution, when the thrust of the air screw is zero, is 1.68 ft.

The curves of Fig. 7 give the relationship between the torque and the translational speed at constant values of the thrust. The value of the thrust and torque of Fig. 1 and 7 are to be used for the steel and atmosphere of temperature the degree F and pressure 14.7 lb. sq. in. The curves have been calculated from the observations of temperature and pressure taken during the experiments.

The curves of Fig. 6 and 7, which can be plotted directly from experimental observations, are not in comparison with the results made with other air screws. A small non-constant torque of the air screw is due to the air screw being in a level of absolute rest. An absolute coefficient has the great advantage that its value is independent of the nature of the air screw, and is, in fact, a constant. The air screw is, in fact, a constant. The air screw is, in fact, a constant.

Another feature of the air screw is the fact that the air screw is, in fact, a constant. The air screw is, in fact, a constant. The air screw is, in fact, a constant.

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3 ft. 6 in. In a similar manner we may write either $Q = Q_0 \sqrt{V/V_0}$ or $Q = Q_0 \sqrt{V/V_0}$. In Fig. 8 the performance of the air screw has been compared in two ways, first, by plotting T , and Q , against $(\frac{V}{V_0})^2$, and second, by plotting T , and Q , against $(\frac{V}{V_0})^2$, $(\frac{V}{V_0})^2$ being also at absolute coefficient. Since $T_0 = T$, $(\frac{V}{V_0})^2$ and $Q_0 = Q$, the air screw performance may be

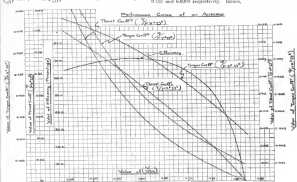


FIG. 8. PERFORMANCE CURVES OF AN AIR SCREW

expressed completely by plotting either T , and Q , against $(\frac{V}{V_0})^2$, or T , and Q , against $(\frac{V}{V_0})^2$.

The difference in the ratio of the useful work done by the air screw to the total work put into the air screw is the efficiency.

$$\eta = \frac{\text{Thrust} \times \text{Translational speed}}{2\pi \times \text{Torque} \times \text{Rotational speed}} = \frac{T}{2\pi Q} \times \frac{V}{V_0} \times \frac{V_0}{V} = \frac{T}{2\pi Q}$$

Efficiency being the ratio of two like quantities η , of course, is a dimensionless coefficient. We see, then, that the performance of an air screw may be expressed completely in two dimensionless coefficients—namely

$$\left(\frac{T}{2\pi Q} \right) \text{ and } \left(\frac{V}{V_0} \right) \text{ or } \left(\frac{T}{2\pi Q} \right) \text{ and } \left(\frac{V}{V_0} \right)$$

and the coefficient of such a system will be apparent from a dimensionless such as in the following paragraph.

To calculate the values of the thrust and torque of the air screw at ground level, when it is moving through the air with a translational speed of 55 m.p.h. and a rotational speed of 200 r.p.m., we express the known data in convenient units, and

$$\begin{aligned} V &= 150 \text{ ft. per sec.} \\ \omega &= 20 \text{ rev. per sec.} \\ D &= 0.347 \text{ ft.} \end{aligned}$$

$\rho = 0.00237$ slugs per cu. ft. (assuming the temperature of the air to be 15.5 deg. Cent. and the pressure 760 mm.)

The value of $(\frac{V}{V_0})^2$ at which the air screw is working is $(\frac{150}{55})^2 = 7.38$, that is 0.06, the corresponding values of $(\frac{T}{2\pi Q})$ and $(\frac{V}{V_0})$ are measured from the curves being 0.157 and 0.6208 respectively. Hence,

$$\begin{aligned} T &= 0.00237 \times (0.157)^2 \times (1160)^2 \times 0.347 = 379 \text{ lb.} \\ Q &= 0.00237 \times (0.6208)^2 \times (1160)^2 \times 0.347 = 400 \text{ lb. ft.} \end{aligned}$$

$$\eta = \frac{T}{2\pi Q} = \frac{379}{2\pi \times 400} = 0.30$$

The thrust and torque could, of course, be calculated from the values of $(\frac{T}{2\pi Q})$ and $(\frac{V}{V_0})$.

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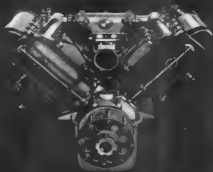
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